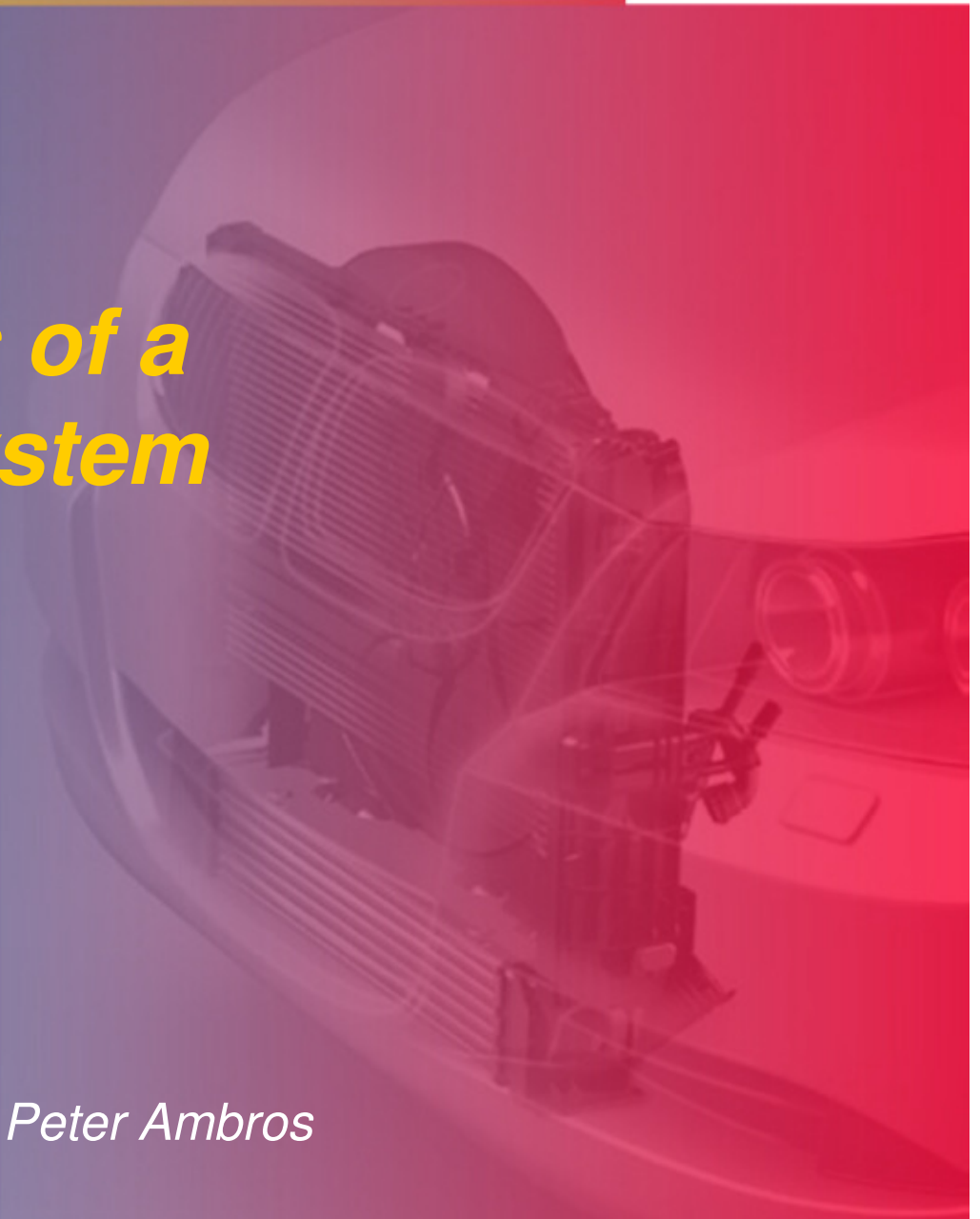


# ***Thermal Analysis of a Truck Cooling System***

*KULI User Meeting  
June 14, 2007*

*Stefan Hildinger, Dominik Zurek, Peter Ambros*



## **1. Motivation & Objective**

## **2. Approach**

## **3. Wind Tunnel Measurements**

Test Vehicle & Measurement Program

Measurement equipment

## **4. Simulation**

Thermal Vehicle Simulation Model

Results

## **5. Summary**

## **1. Motivation & Objective**

## **2. Approach**

## **3. Wind Tunnel Measurements**

Test Vehicle & Measurement Program

Measurement equipment

## **4. Simulation**

Thermal Vehicle Simulation Model

Results

## **5. Summary**

## Motivation for development of enhanced simulation model

- Quantify the potential advantage of new cooling system concept on:
  - fuel consumption
  - Cold start and warm-up phase
- increasing complexity of cooling systems
- control of auxiliary units

## Objectives

Enhancement of measurement equipment for vehicle tests

Enhancement of Modine simulation tools

## Objective 1: Enhancement of measurement equipment

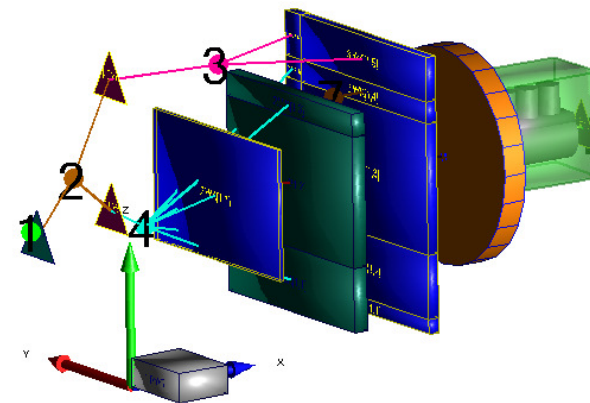
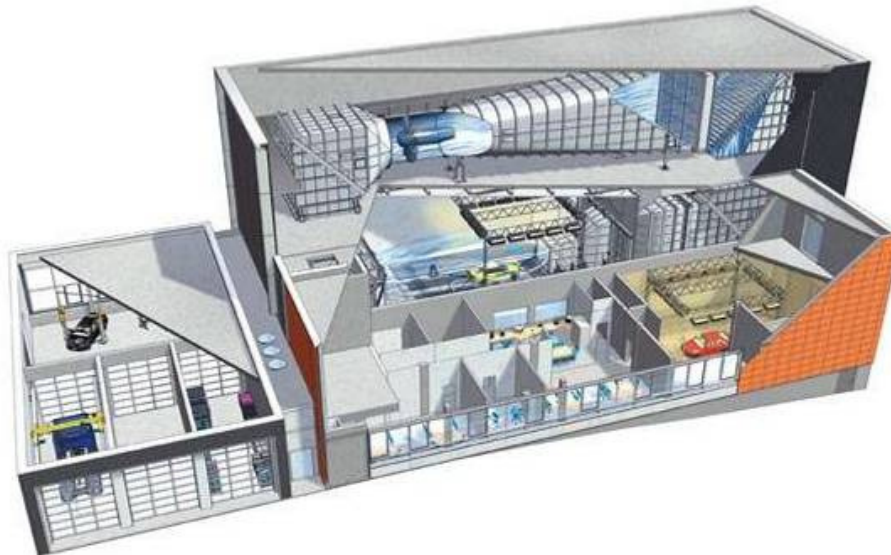
- Equip a test vehicle with measurement instrumentation
- Qualify new measurement equipment in wind-tunnel tests

Measurement

## Objective 2: Enhancement of simulation tools

- Develop an enhanced thermal vehicle simulation model
- Compare simulation results of enhanced model to wind-tunnel measurement

Simulation



## 1. Motivation & Objective

## 2. Approach

## 3. Wind Tunnel Measurements

Test Vehicle & Measurement Program

Measurement equipment

## 4. Simulation

Thermal Vehicle Simulation Model

Results

## 5. Summary

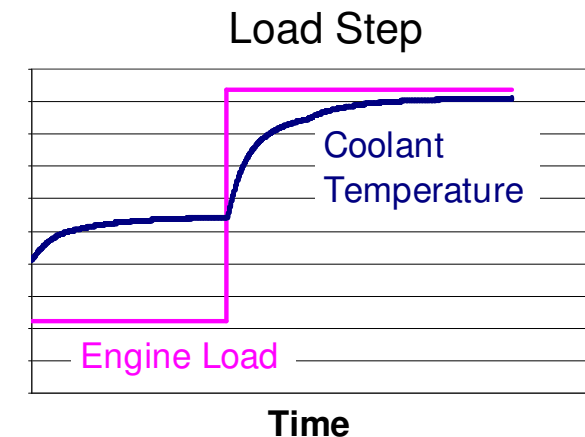
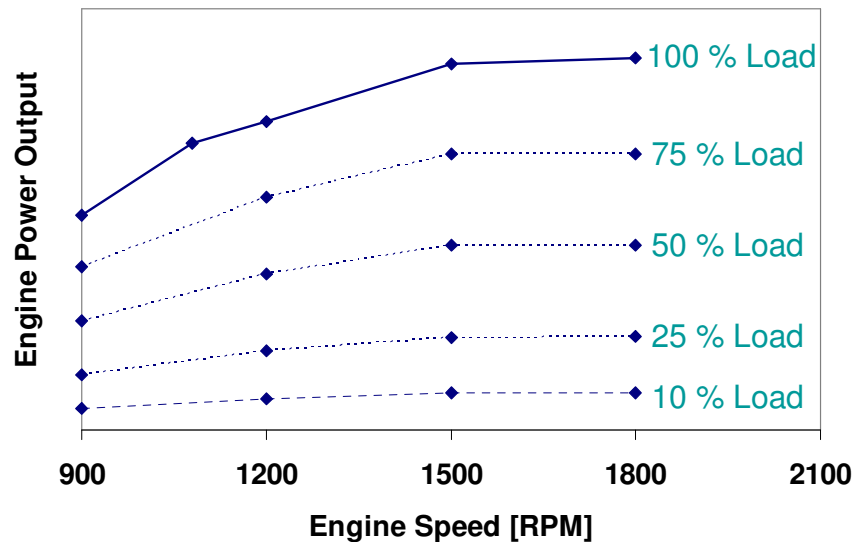
## Test Vehicle

- Truck, V8-engine, 570 hp
- thermostat blocked (open position)
- locked fan clutch

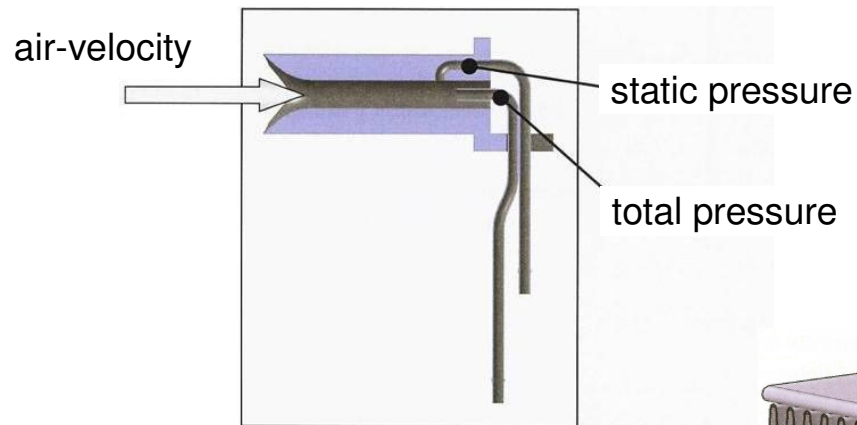


## Measurement program

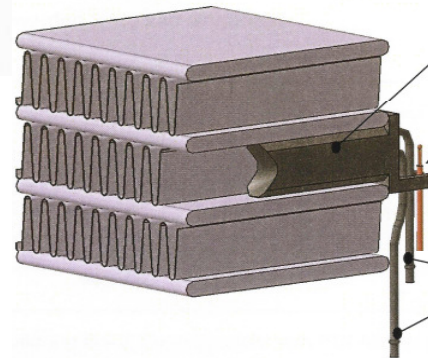
- steady-state operating points  
 $T_{amb}=25^{\circ}\text{C}$ , wind speed 0...135 km/h
- Load step



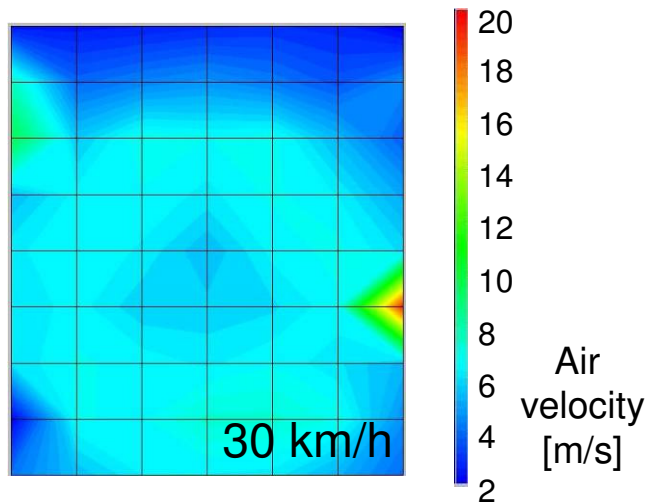
## Measurement of cooling air mass flow rate



Micro-Probe



Installation in core



Radiator equipped with 63 Micro-Probes



## 1. Motivation & Objective

## 2. Approach

## 3. Wind Tunnel Measurements

Test Vehicle & Measurement Program

Measurement equipment

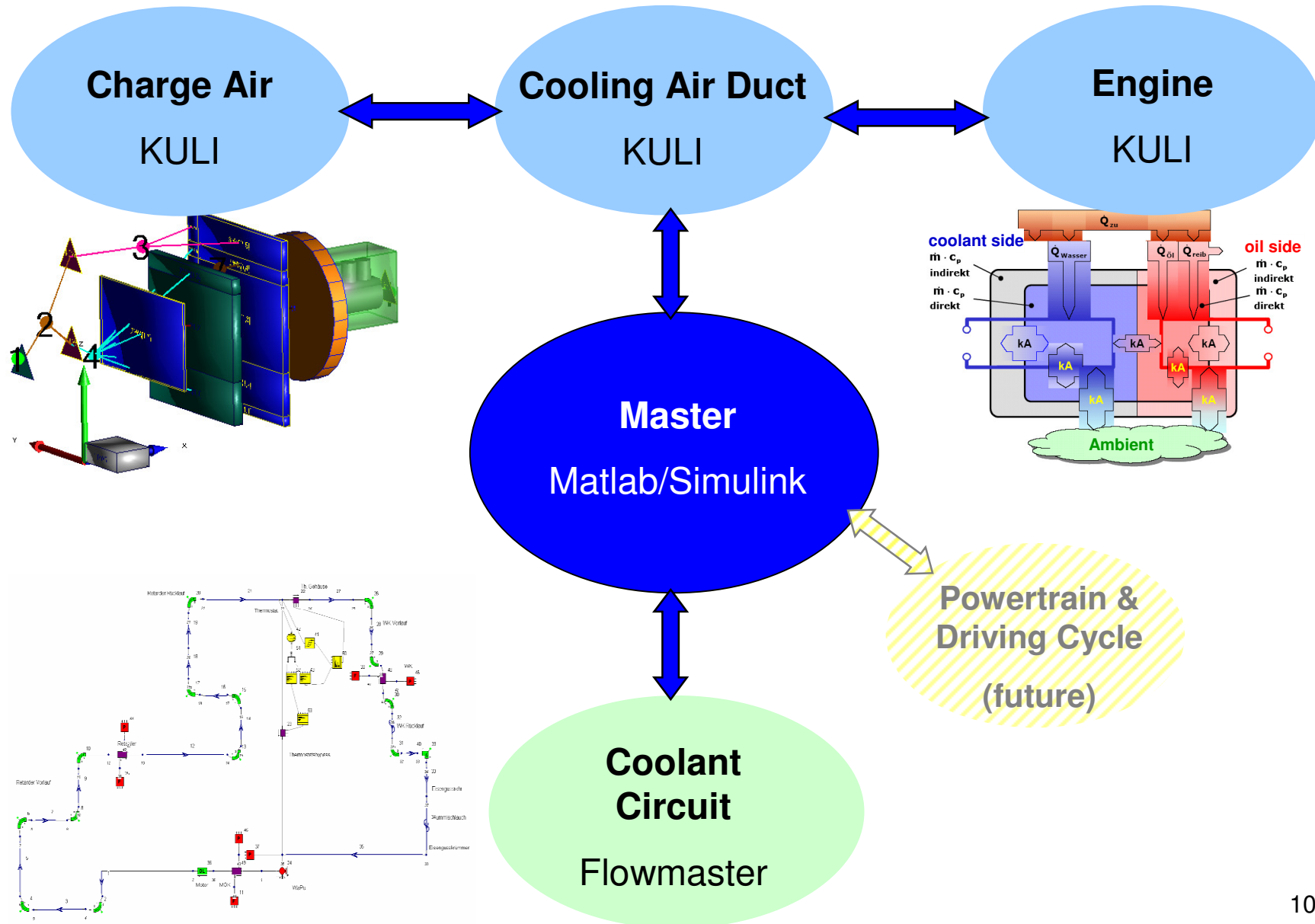
## 4. **Simulation**

Thermal Vehicle Simulation Model

Results

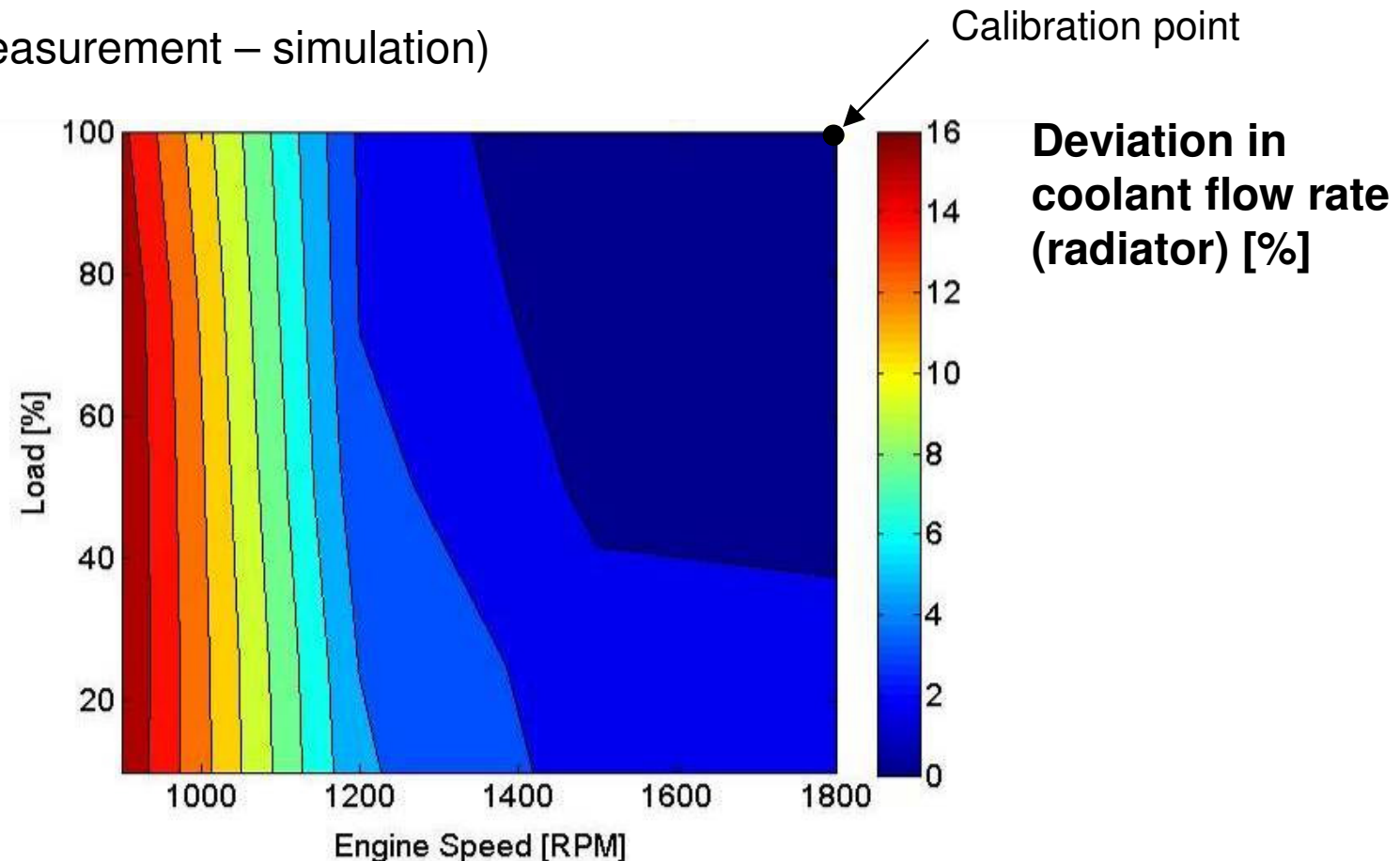
## 5. Summary

# Enhanced Simulation Model



### Coolant: Accuracy of flow rate simulation

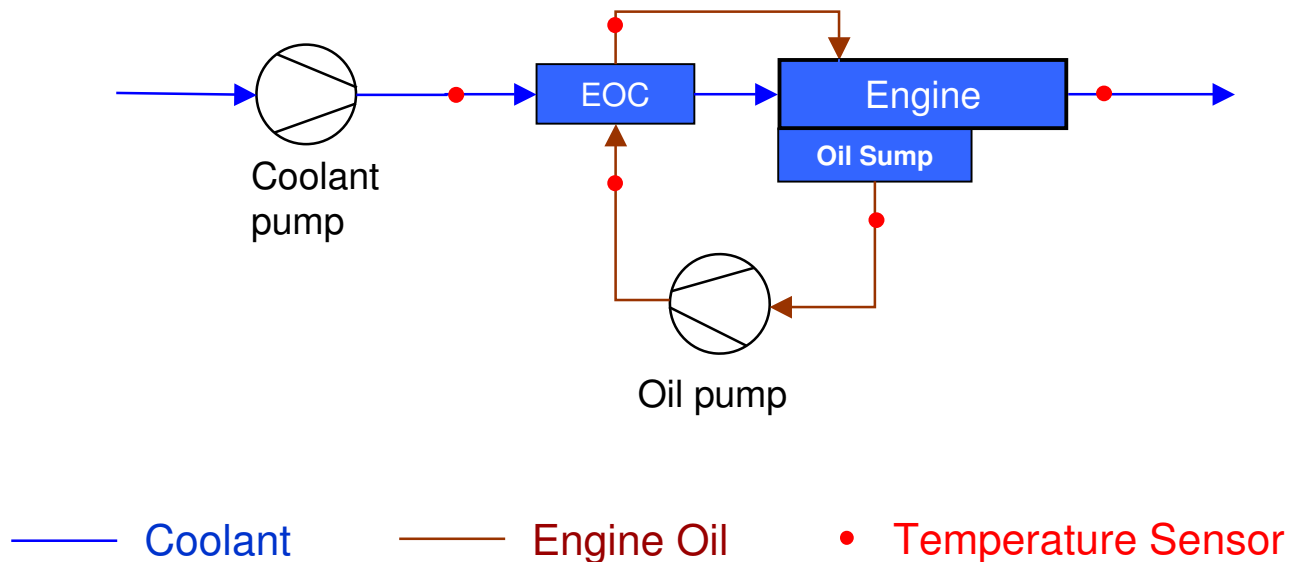
Deviation (measurement – simulation)



- ⇒ deviation < 5% for usual operating range of a truck
- ⇒ higher deviation at engine speeds lower than 1200 RPM due to extrapolation in pump characteristic

## Data for Engine Model

Measurement of coolant and oil temperature

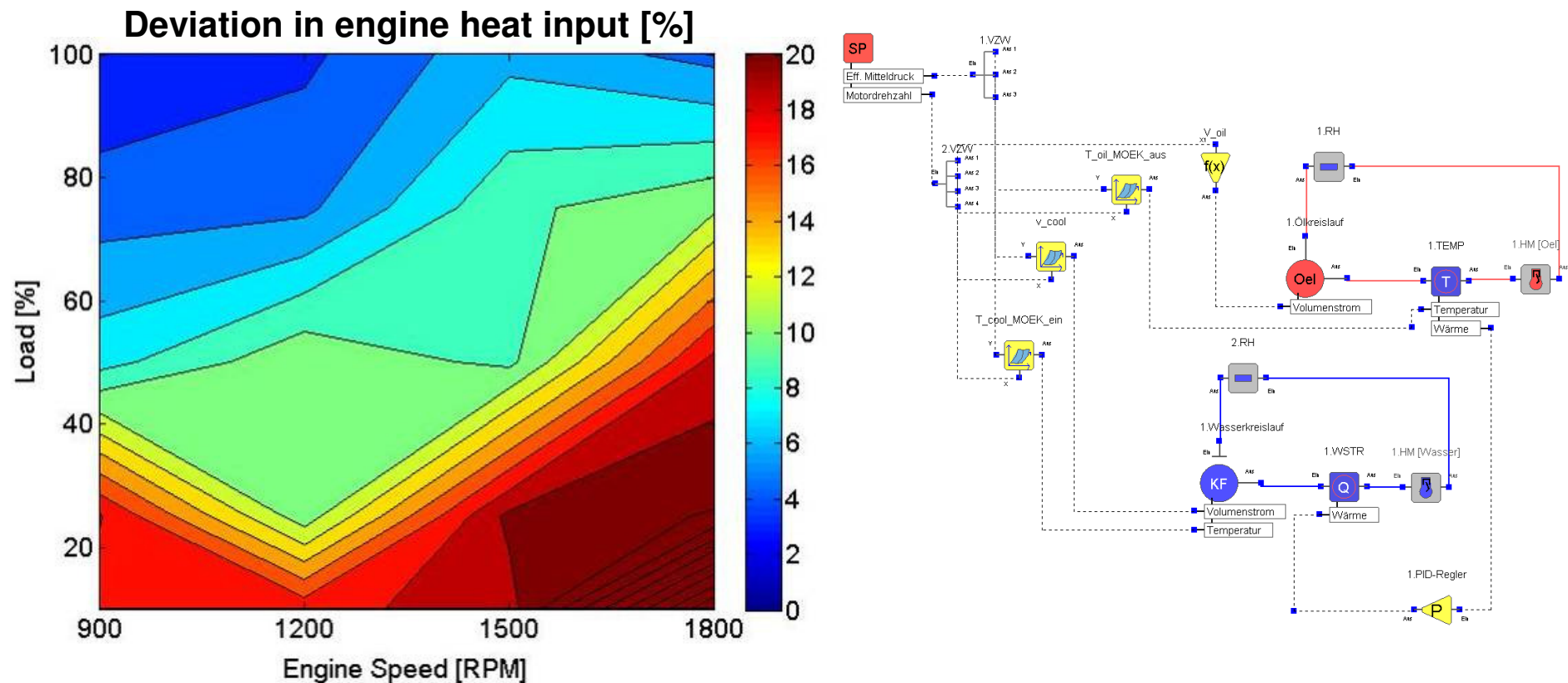


Coolant temperature between engine and engine oil cooler (EOC) could not be measured

⇒ Determination of temperature from heat balance at EOC

## Engine: Accuracy of heat input

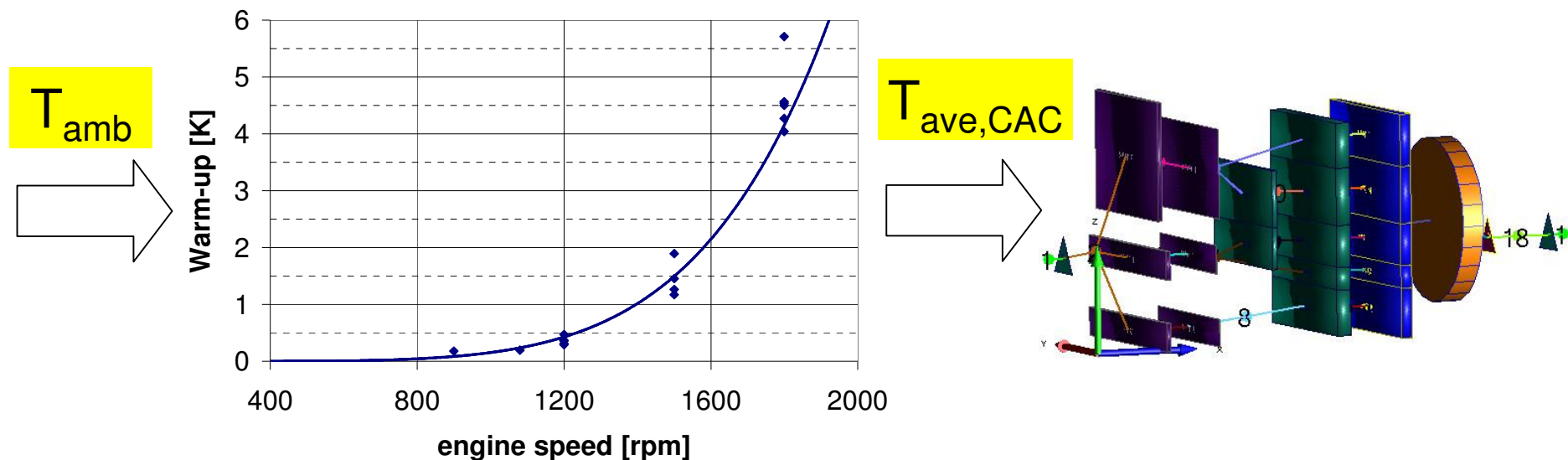
Deviation (measurement – simulation); KULI-engine model built by Magna Powertrain



- ⇒ deviation < 14% for usual operating range of a truck
- ⇒ maximum deviation 17% (i.e. 0.7 K difference in coolant temperature)

## Measured cooling air temperature increase resulting from air recirculation @ full load

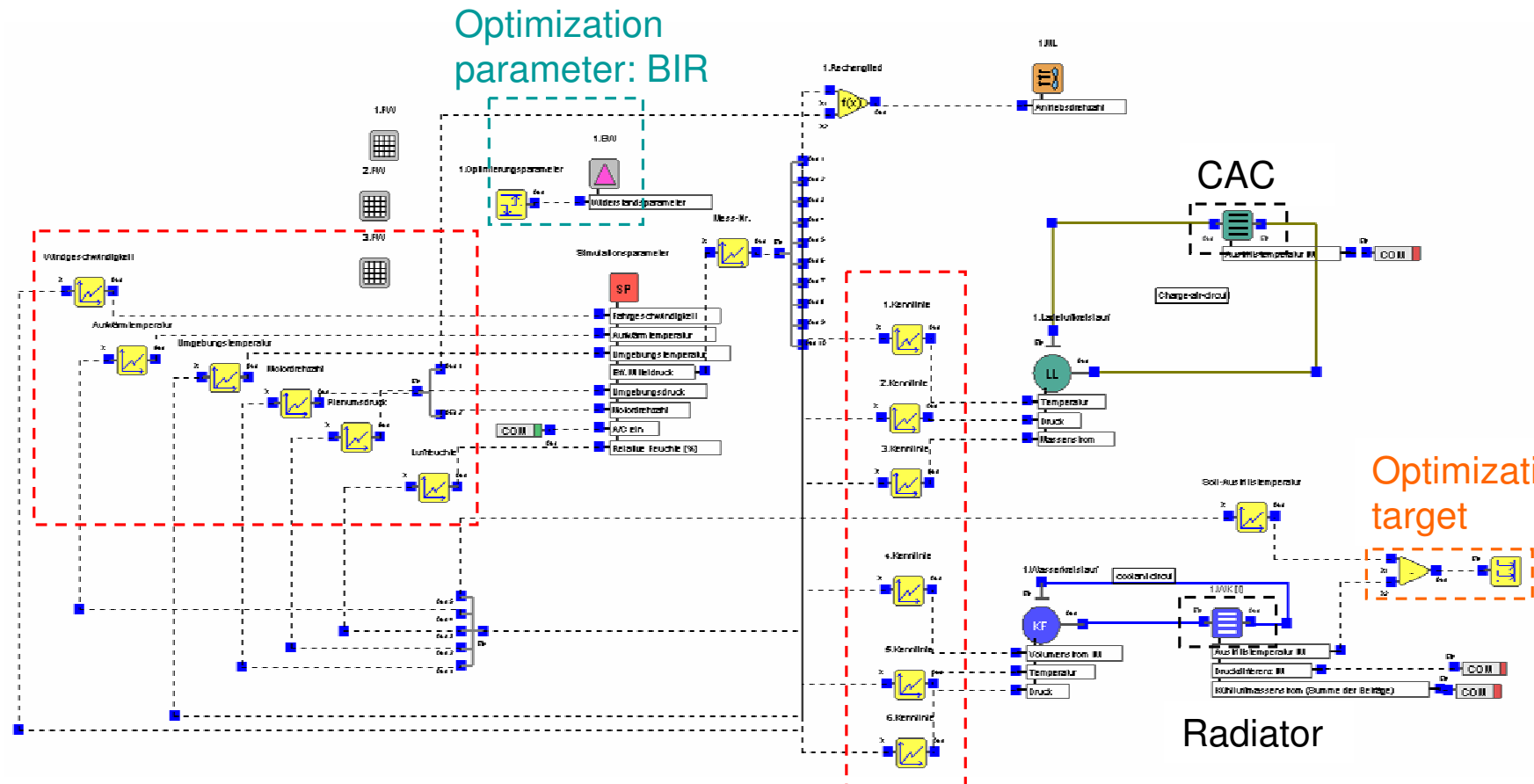
Wind speed 30 km/h



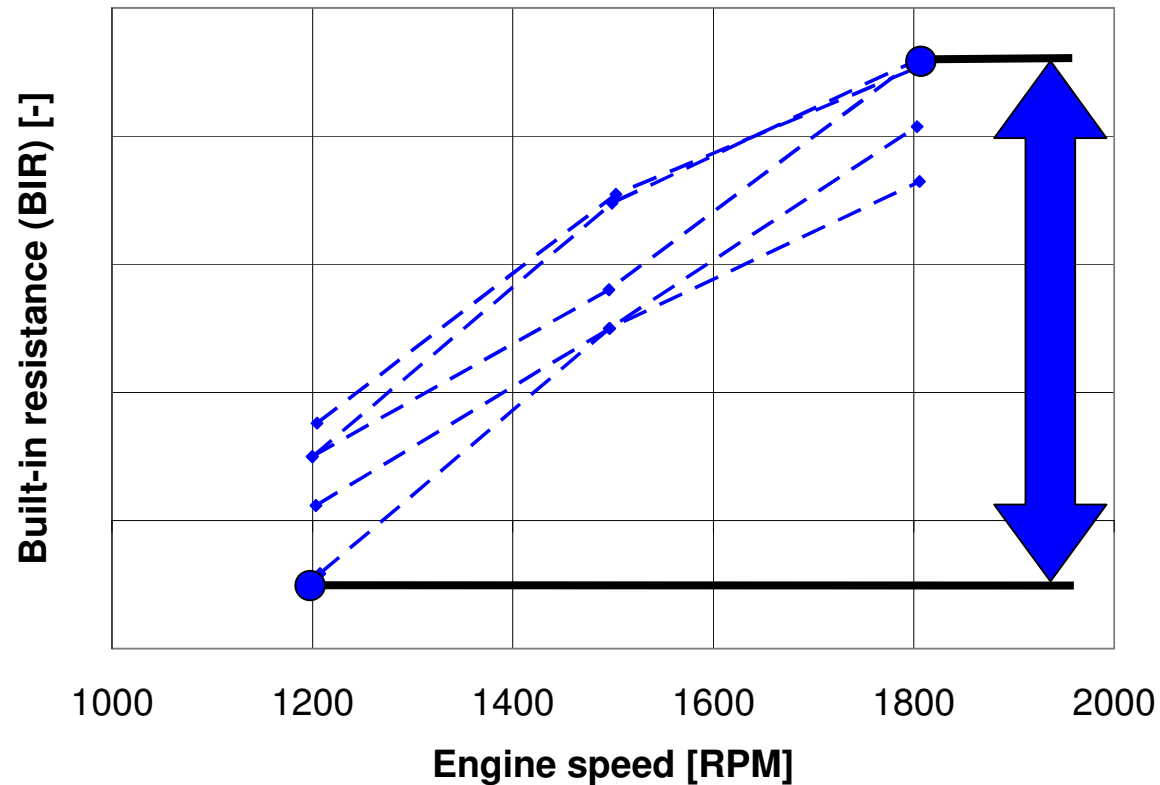
$$\text{Warm-up} = T_{ave,CAC} - T_{amb}$$

- ⇒ strong influence of fan speed
- ⇒ warm-up temperature 4.5 K at 2340 RPM fan speed (=1800 RPM engine speed)

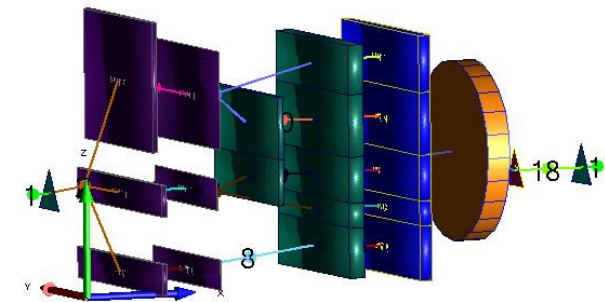
## Calibration of simulation model



⇒ Result: Appropriate built-in resistances for each operating point

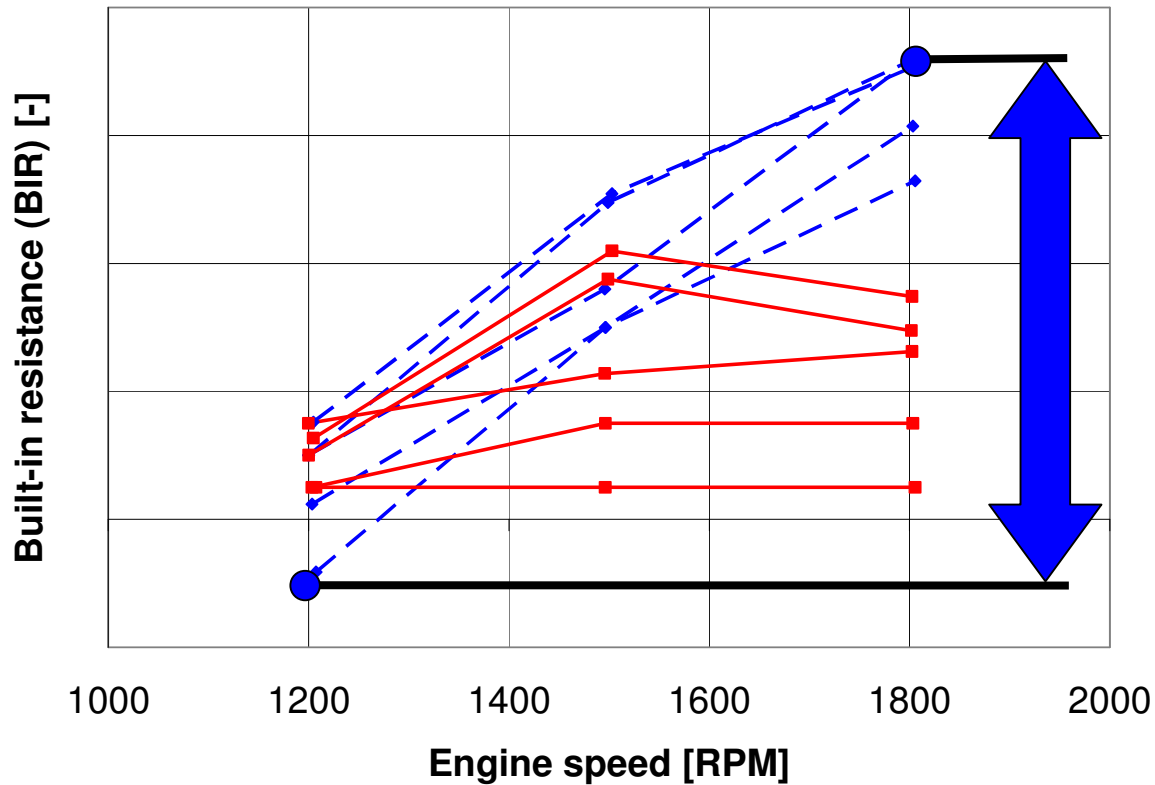


No air recirculation  
(no preheating)



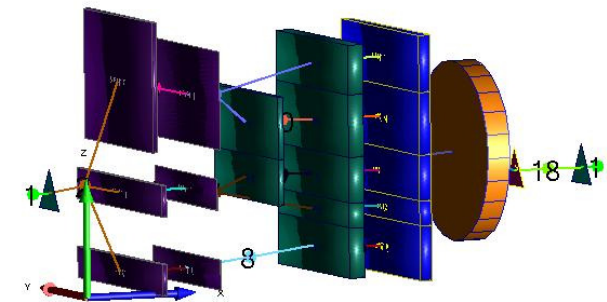
BIR = drag coefficient of cooling air flow through the engine compartment



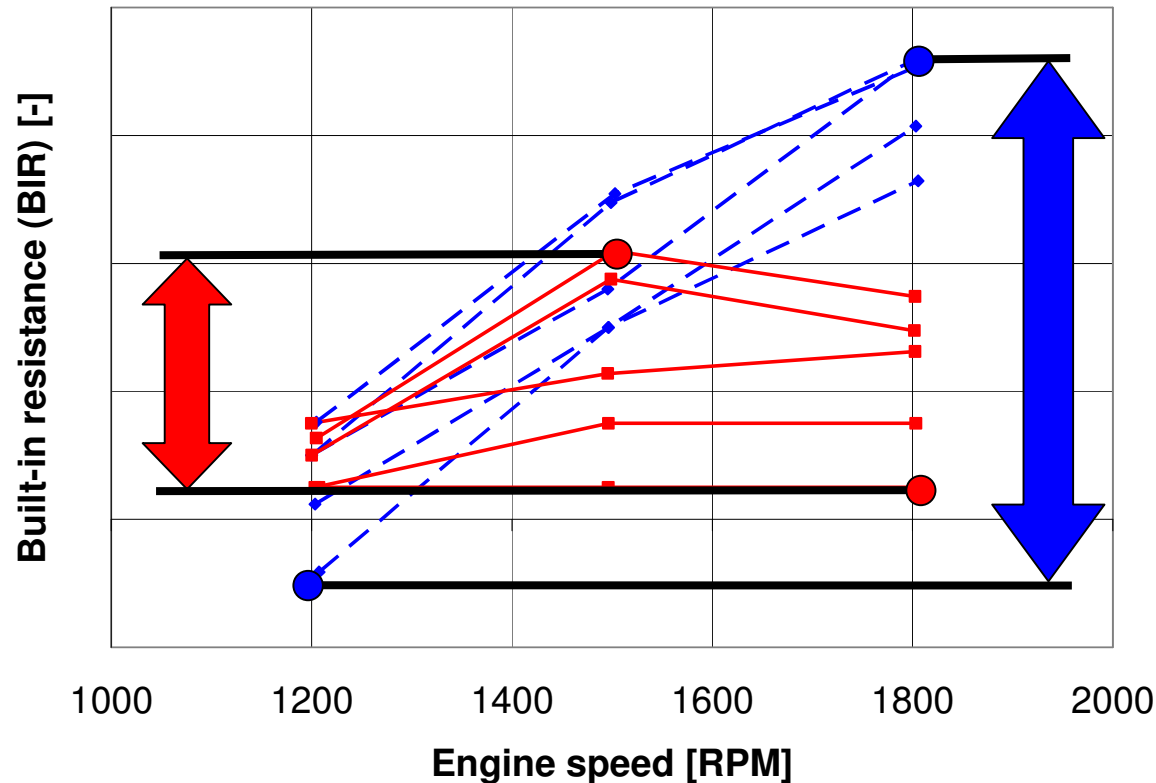


**No air recirculation  
(no preheating)**

**With air recirculation  
(with preheating)**

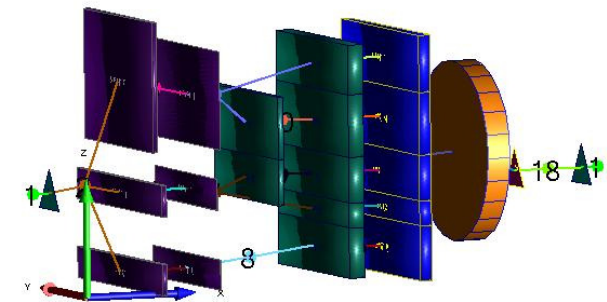


BIR = drag coefficient of cooling air flow through the engine compartment



No air recirculation  
(no preheating)

With air recirculation  
(with preheating)



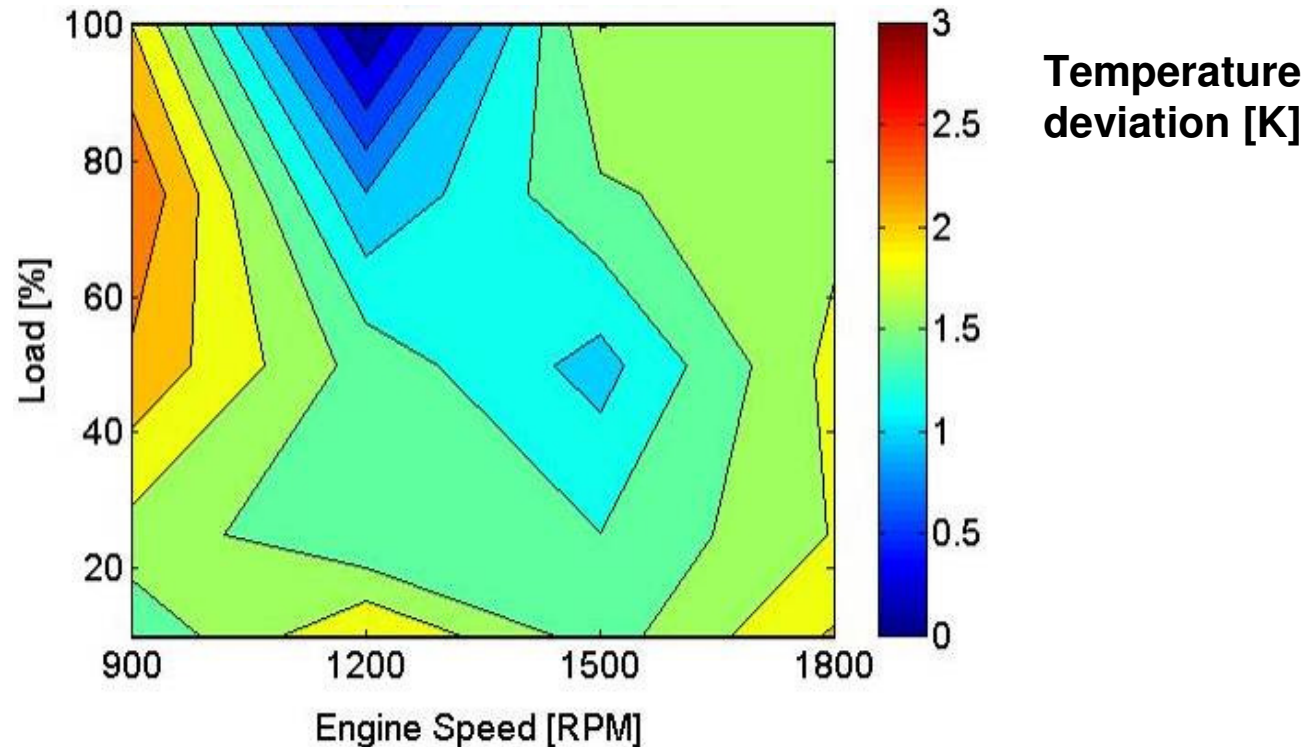
BIR = drag coefficient of cooling air flow through the engine compartment

With air recirculation:

- ⇒ Smaller range of BIR
- ⇒ Significantly lower deviation of simulation results in case only average BIR is available

### Coolant: Accuracy of temperature simulation

Deviation (measurement – simulation)



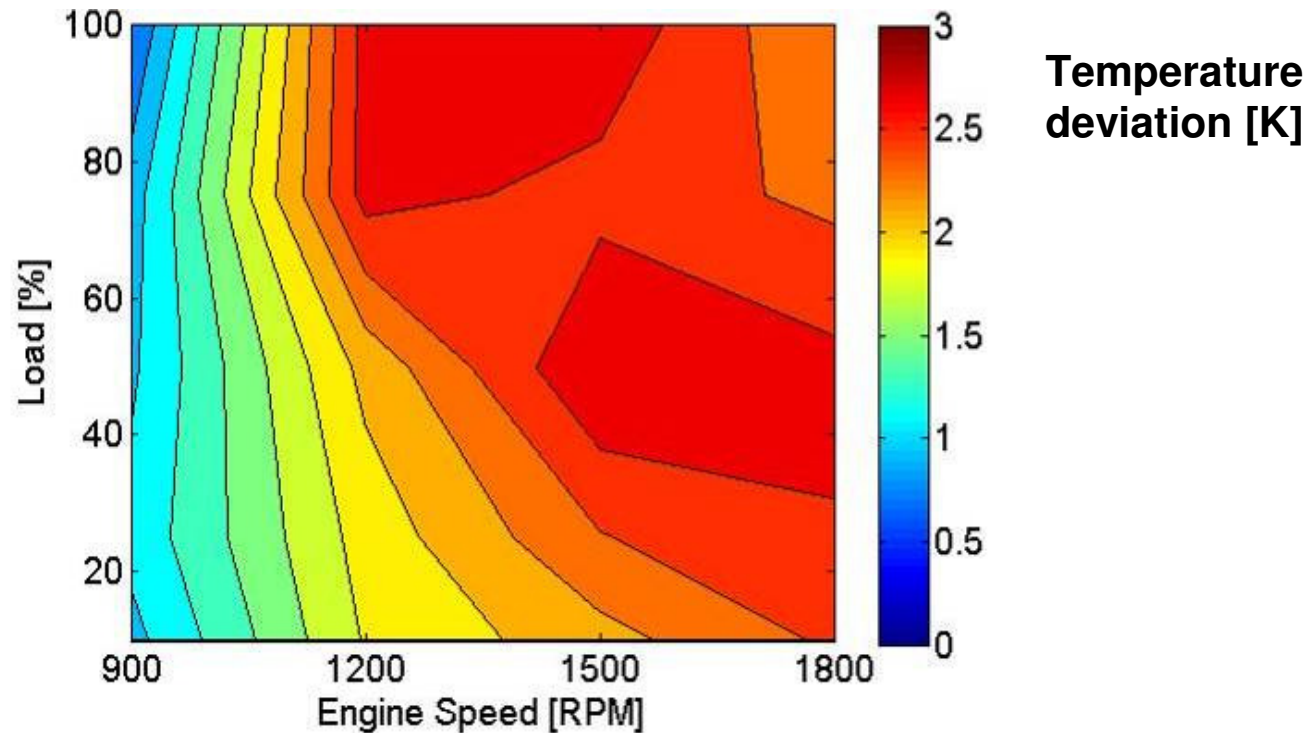
Deviation in flow rate, engine heat input and cooling air side summarized in coolant temperature:

⇒ Deviation in coolant temperature < 2.5 K

⇒ Deviation in coolant temperature < 1.5 K for usual operating range of a truck

### Charge-Air: Accuracy of temperature simulation

Deviation (measurement – simulation)



- ⇒ Charge-Air temperature not calibrated to measurement
- ⇒ Deviation in Charge-Air temperature shows accuracy of model
- ⇒ Deviation in Charge-Air temperature < 3K !

## 1. Motivation & Objective

## 2. Approach

## 3. Wind Tunnel Measurements

Test Vehicle & Measurement Program

Measurement equipment

## 4. Simulation

Thermal Vehicle Simulation Model

Results

## 5. Summary

## Measurement

- Qualification of new measurement technique
- Quantification of pre-heating of cooling-air determined: up to 5 K

## Thermal vehicle simulation model

- Simulation model built up to handle interactions between engine heat load, coolant flow, charge air and cooling air flow
- Complete vehicle operating range can be simulated
- Improvement of simulation accuracy by consideration of hot air-recirculation
- Accuracy of charge-air and coolant temperature simulation better than 3 K for complete operation range

## Overall

- Measurement technique and simulation model in use for research & application projects

**Thank you!**